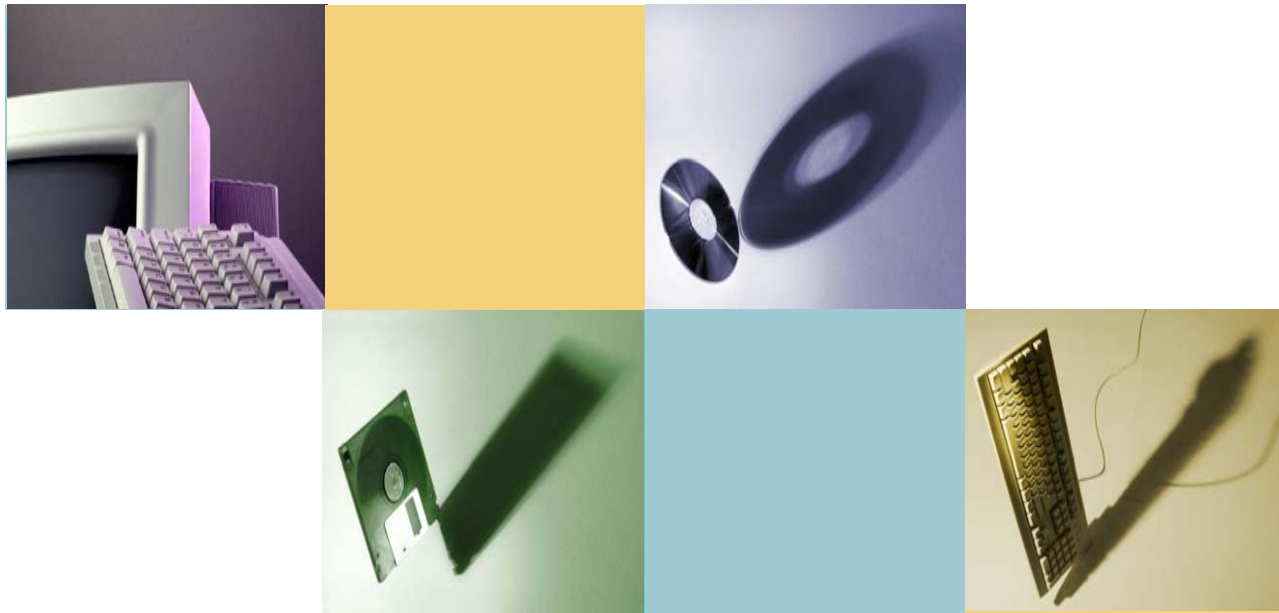


Room cooling control optimization



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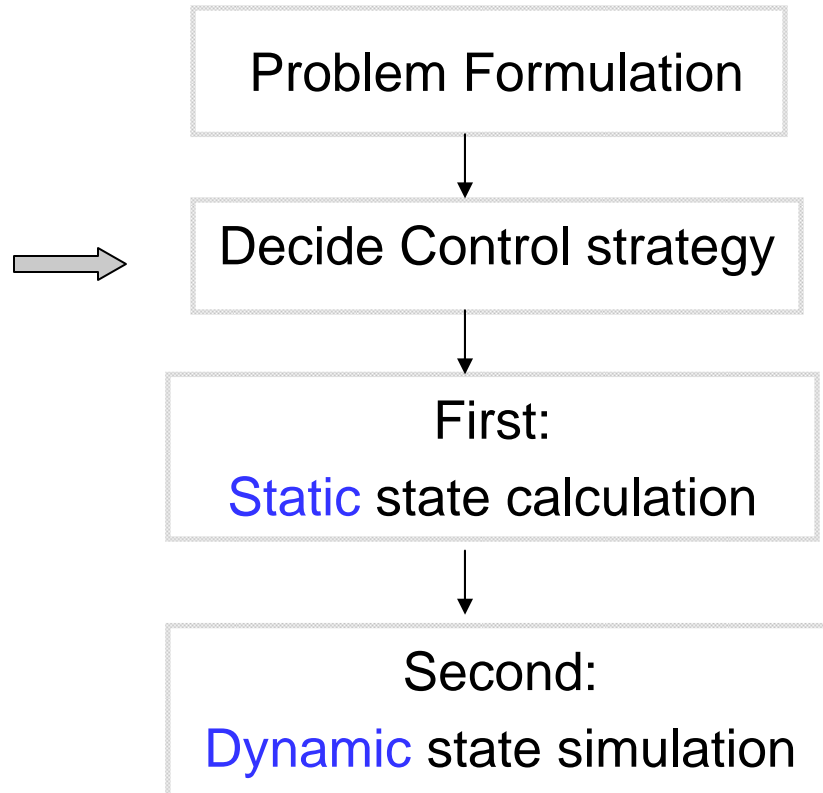
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Overview

- Apply the optimization method to control the building thermal environment
- Want to maintaining thermal comfort and minimizing the energy cost



Functions & Parameters

- Thermal comfort index
 - PMV
 - 'predicted mean vote' on the thermal sensation scale
 - $PMV = (-8.6479 + 0.2431 \cdot C) + (0.3422 - 0.0073 \cdot C) \cdot T$
 - C: Relative humidity
 - T: Air temperature
 - PPD
 - the predicted percentage of dissatisfied people
 - $PPD = 100 - 95 \cdot \exp[-\{0.03353 \cdot PMV^4 + 0.2179 \cdot PMV^2\}]$

Functions & Parameters

- Heat balance equation

$$a \cdot C_{\text{air}} \cdot V_{\text{room}} \cdot (T_{p+1} - T_p) / \Delta t = \sum h \cdot A_i \cdot (T_p - T_{si}) + Q_1 + Q_2$$

a: Air density of room [kg/m³]

C_{air}: Specific heat of the air [J/kg°C]

V_{room}: Volume of room (m³)

T_{p+1}: The room air temperature (°C)

T_p: The room air temperature before Δt second (°C)

T_s: The surface temperature of room (°C)

Δt: Time step(sec)

h_i: heat transfer coefficient of wall i (Kcal/m²h°C)

Q₁: cooling load by infiltration (W)

Q₂: generated heat in room (W)

A_i: area of wall i (m²)

$h_i = 2.63^{**} (T_{si} - T_{ip})^{(1/4)}$

Problem Formulation

Objective function: Multi objective nonlinear

- ✓ Min $f_1 + f_2$
- ✓ $f_1 = V_i * (T_{i,p+1} - T_{i,p}) / \Delta t$
- ✓ $f_2 = 100 - 95 * \exp[-\{0.03353 * PMV_i^4 + 0.2179 * PMV_i^2\}]$
- ✓ $PMV = (-8.6479 + 0.2431 * C_i) + (0.3422 - 0.0073 * C_i) * T_{i,p+1}$
- ✓ Consider $T_{i,p+1}$ unique control variable for dynamic simulation

Problem variables

$T_{i,p+1}$: The optimal air temperature of room i ($^{\circ}\text{C}$)

$T_{i,p}$: The air temperature of room i before Δt second ($^{\circ}\text{C}$)

Δt : Time step(sec)

C_i : The relative humidity of room i (%)

$T_{s,i}$: The floor surface temperature of room i ($^{\circ}\text{C}$)

A_i : The floor area of room i (m^2)

V_i : The volume of room i (m^3)

Problem Formulation

- Constraints

- Minimum cooling load should be satisfied

- $F1 \geq 2.63 * (T_{si} - T_{ip}) * A_i * (T_{ip} - T_{si}) / (V * 10 / \Delta t)$



- **Linear constraint**

- Maximum thermal discomfort should be 10%

- $F2 \leq 10\%$



- **Nonlinear constraint**

Control Strategy

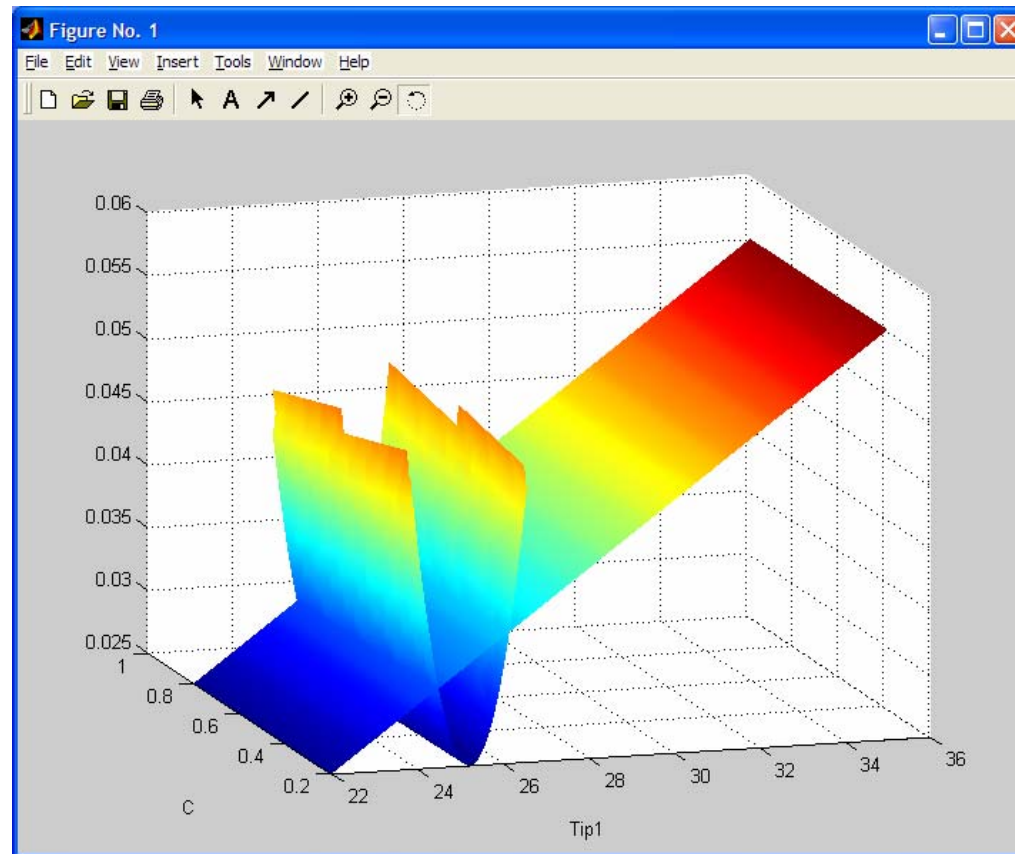
If $T_{i,p+1} < T_{i,p}$ then **ON** the cooling for i th room

If $T_{i,p+1} > T_{i,p}$ then **OFF** the cooling for i th room

```
For i = 1 To N
  IsItOn(i) = WasItOn(i)
  If (Ti,p > (Ti,p+1 - Range)) Then
    IsItOn(i) = True
  Elself (Ti,p < (Ti,p+1 + Range)) Then
    IsItOn(i) = False
  End If
Next i
```

Static State Calculation

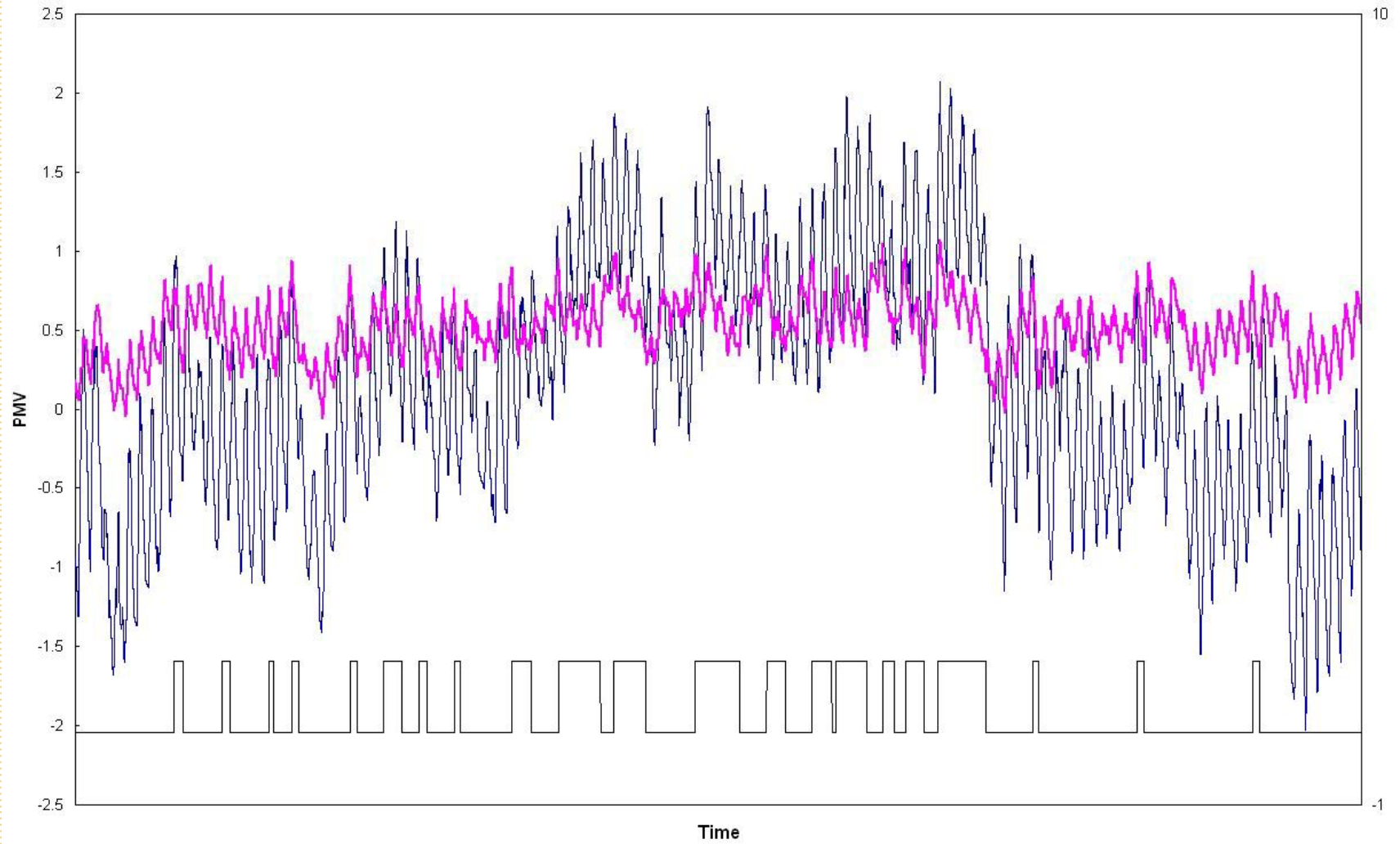
- Ignore Δt (timestep)
- Variable: humidity, temperature



Dynamic State Simulation

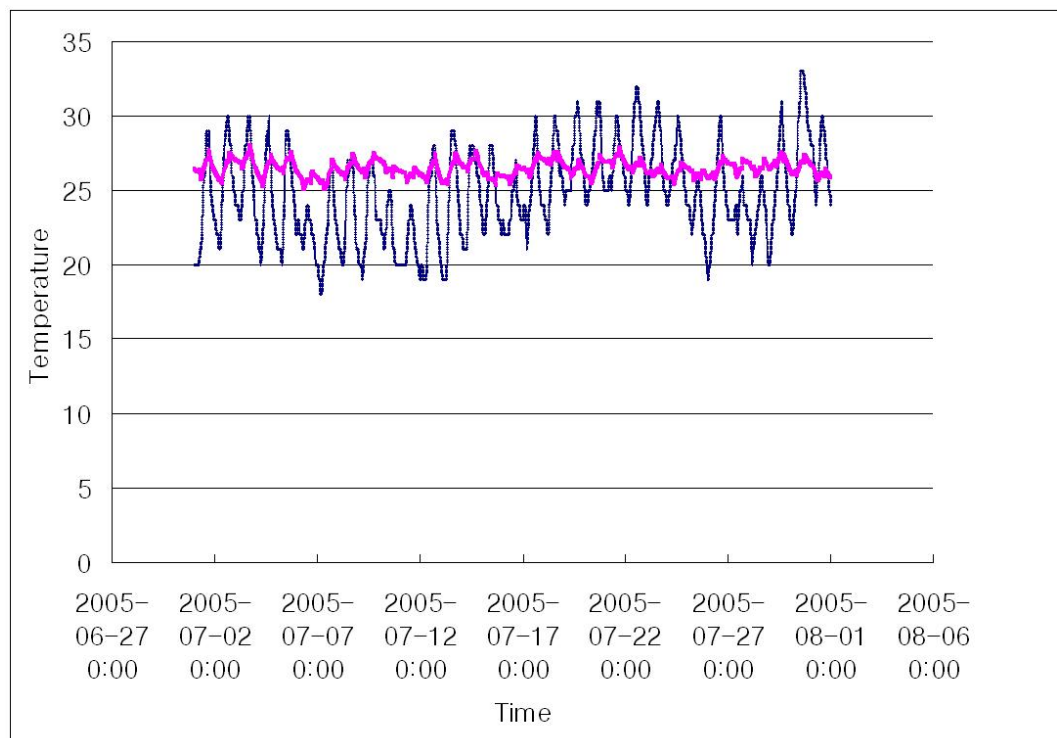
- Link Matlab to VB program for simulation and use function fmincon() for optimization
- Input Data: Weather data, Room modeling
- June~September

Result



Result

- PMV is within the comfort range
- Room thermal environment is well controlled by optimization method



Future Work

- Apply the whole building with multiple rooms
- Examine the effect of the number of sensors to control the building thermal environment
- Consider weighting sum approach
- Compare to other control method



Thank you !
